

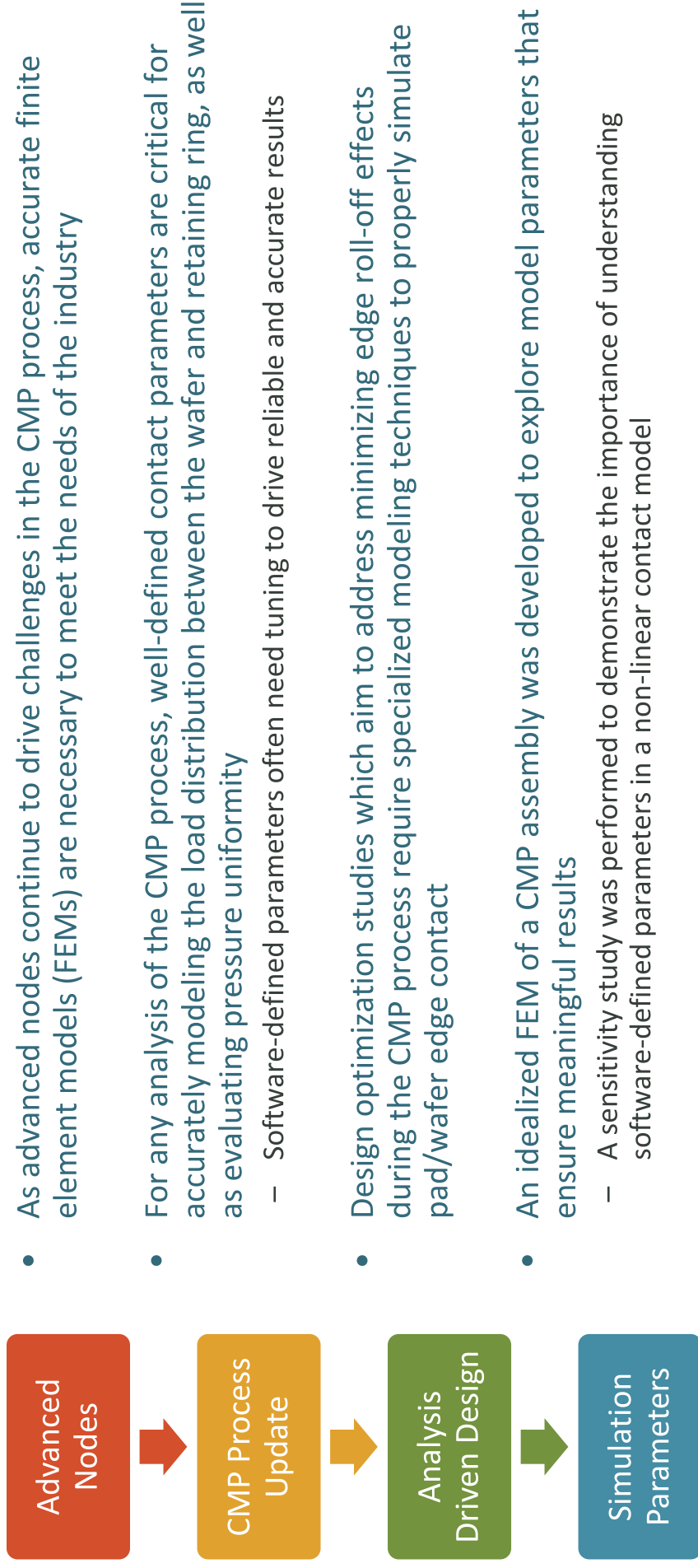
MODEL VERIFICATION TECHNIQUES FOR ANALYSIS OF EDGE ROLL OFF EFFECTS DURING THE CMP PROCESS

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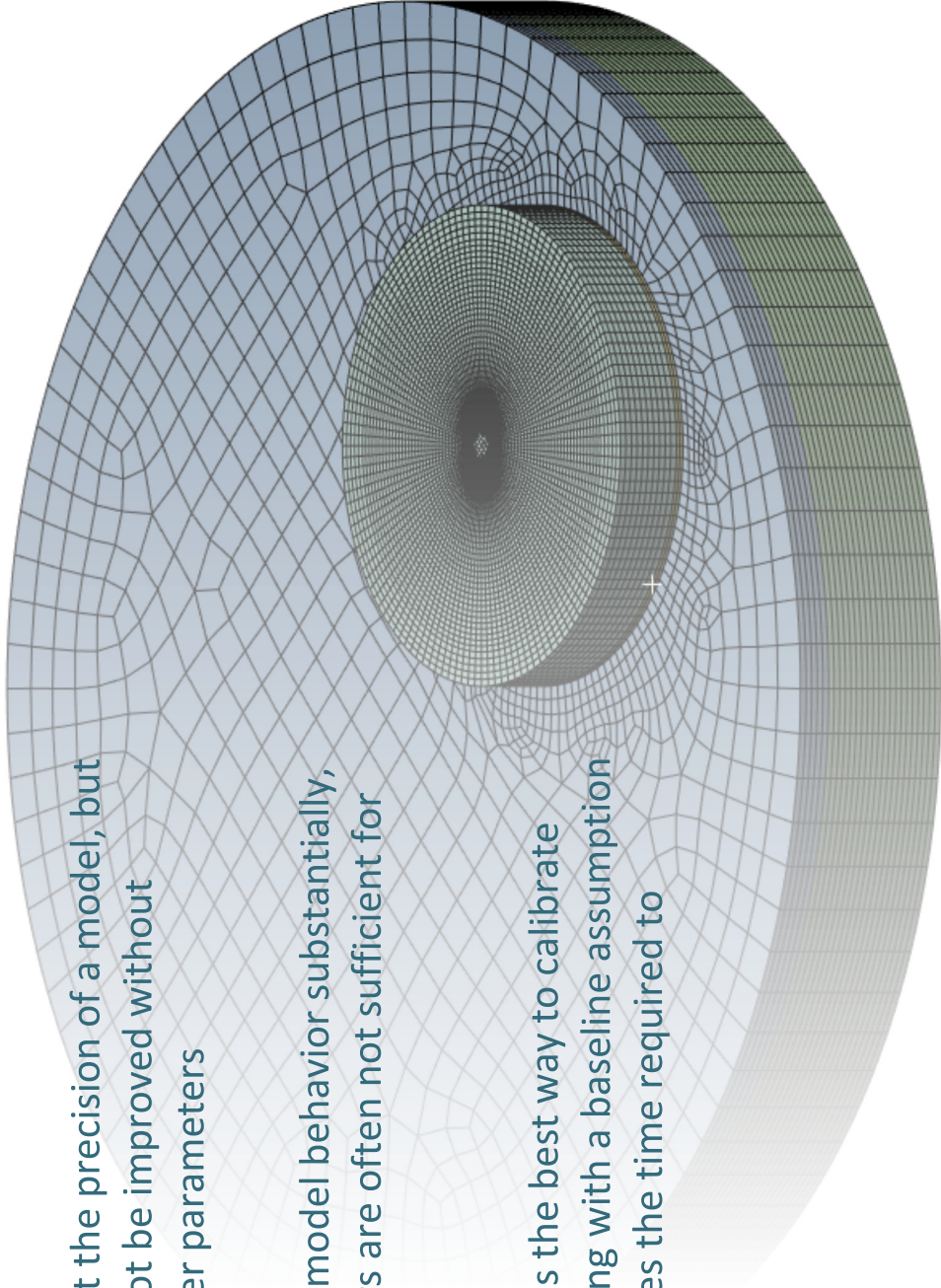
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INTRODUCTION

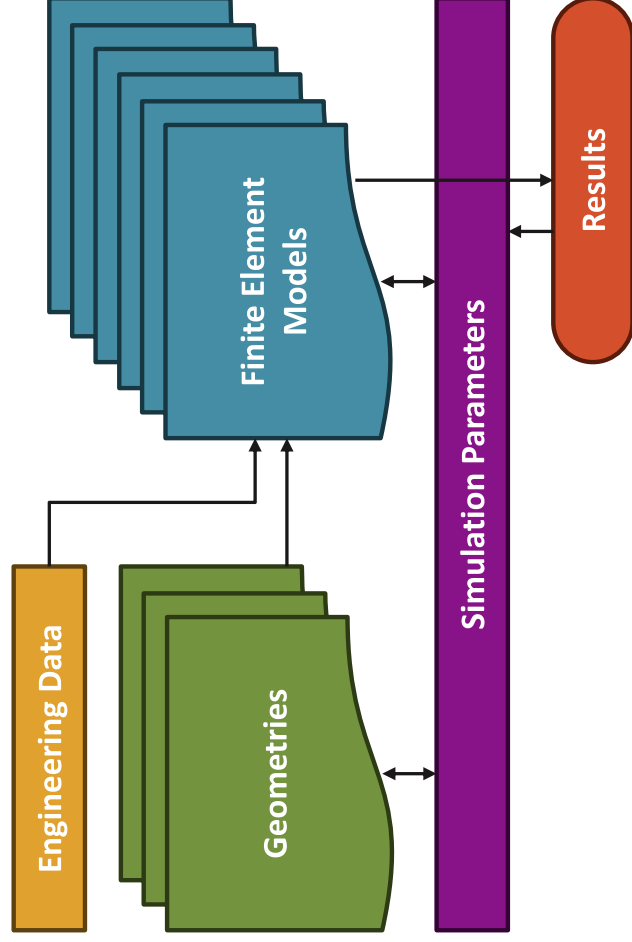


BACKGROUND

- Mesh density studies can impact the precision of a model, but the accuracy of the model cannot be improved without understanding the effect of other parameters
- Contact parameters can impact model behavior substantially, and default software parameters are often not sufficient for generating an accurate model
- Anchoring a model to test data is the best way to calibrate unknown parameters, but starting with a baseline assumption that closer mimics reality reduces the time required to converge on an accurate result



EXPERIMENTAL PROCEDURE



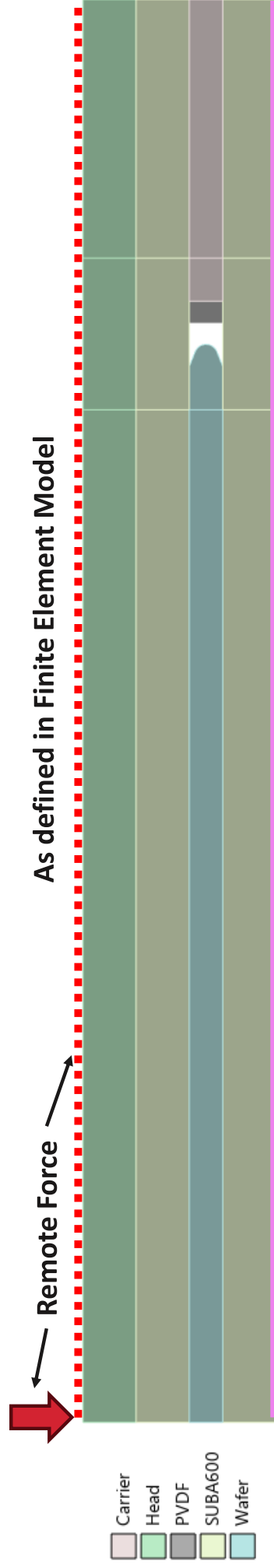
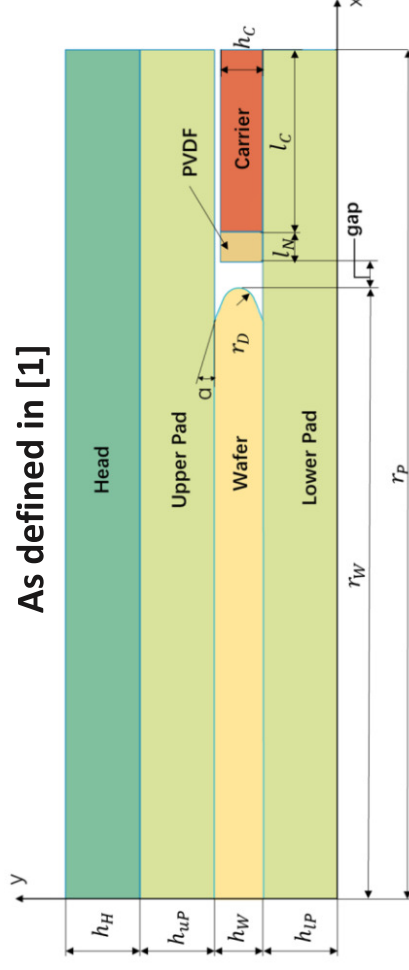
- Reference model generated from paper [1]
- Engineering Data
 - Material properties
- Geometries
 - 2D Axisymmetric and 3D
 - Double sided and single sided configurations
 - Wafer/retainer offsets
- Simulation Parameters
 - Contact stiffness
 - Friction
 - Nodal alignment
- Results
 - Contact pressure distribution
 - Computation time

[1] J. Chen, Y. Liu, D. Wang, W. Yu, and L. Zhu, "Numerical Analysis in Double-Sided Polishing: Mechanism Exploration of Edge Roll-Off," *Materials*, vol. 17, no. 19, p. 4761, Sep. 2024, doi: <https://doi.org/10.3390/ma17194761>.

GEOMETRY, MATERIAL PROPERTIES

- Reference geometry and material properties from [1]

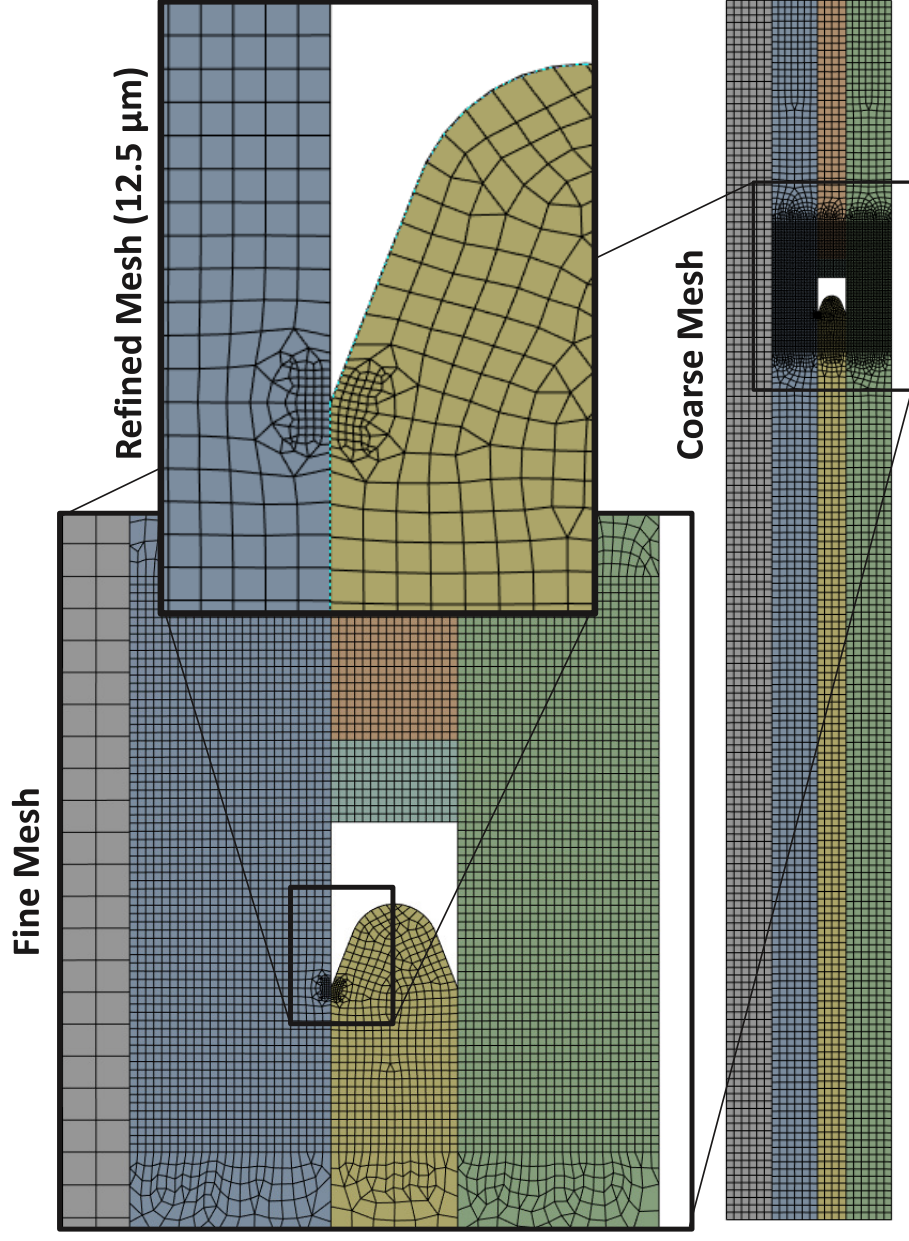
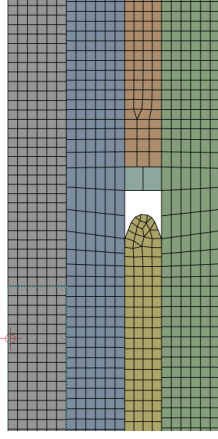
Part	Young's Modulus (MPa)	Poisson's Ratio
Head	5000	0.3
SUBA800M2	205.8	0.2599
SUBA600	34.14	0.2448
SL12	243.25	0.3229
Carrier	193,000	0.31
Wafer	190,000	0.27
PVDF	840	0.3



MESH DEFINITION

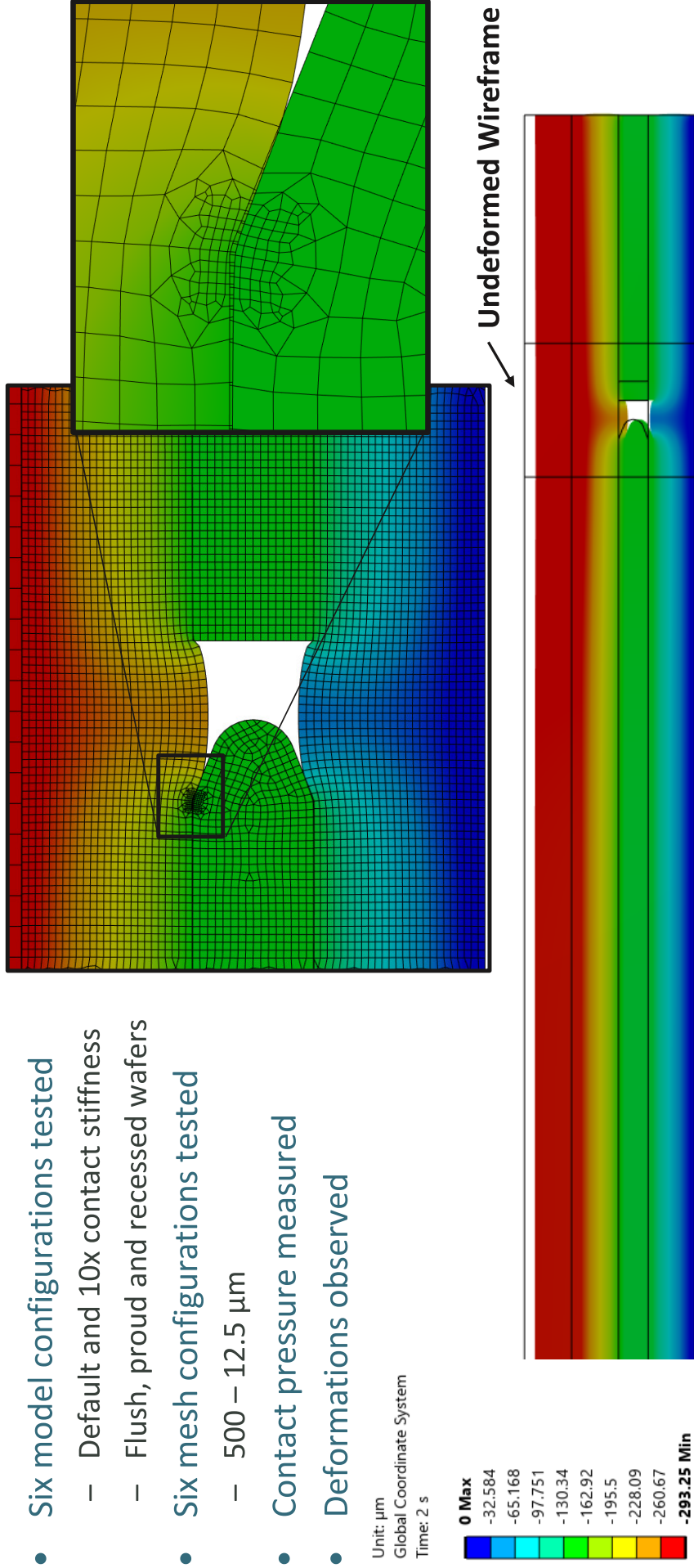
- Coarse mesh used for stiffness representation remote from area of interest
- Fine mesh used around edge roll off locations
- Refined mesh used at corner contact location
- Grid size was not specifically studied or converged upon
- Grid sizes tested: 500 – 12.5 μm

Coarsest Mesh (500 μm)



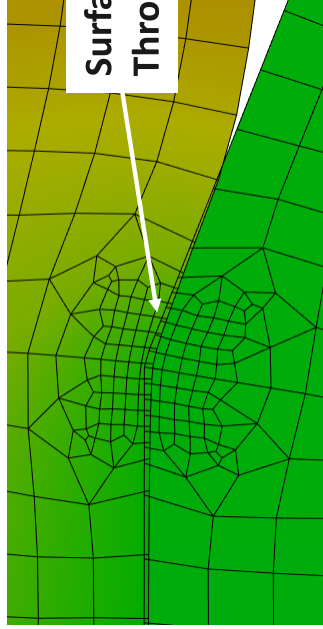
RESULTS | 2D FEM CONTACT STIFFNESS

- Six model configurations tested
 - Default and 10x contact stiffness
 - Flush, proud and recessed wafers
- Six mesh configurations tested
 - 500 – 12.5 μm
- Contact pressure measured
- Deformations observed

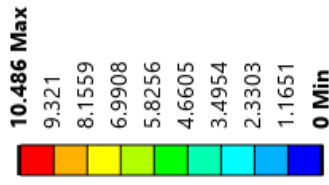


RESULTS | 2D FEM CONTACT STIFFNESS

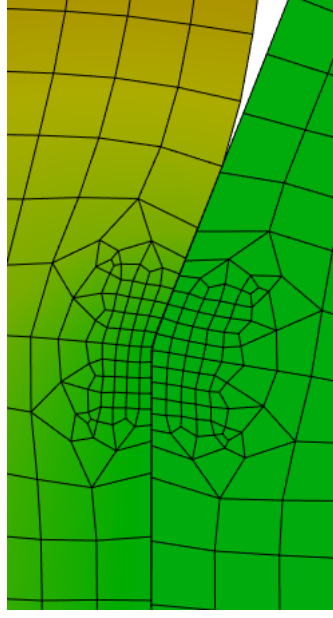
Default Contact Stiffness



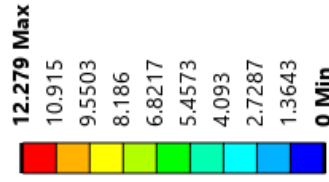
Type: Pressure
Unit: MPa
Time: 2 s



10x Contact Stiffness



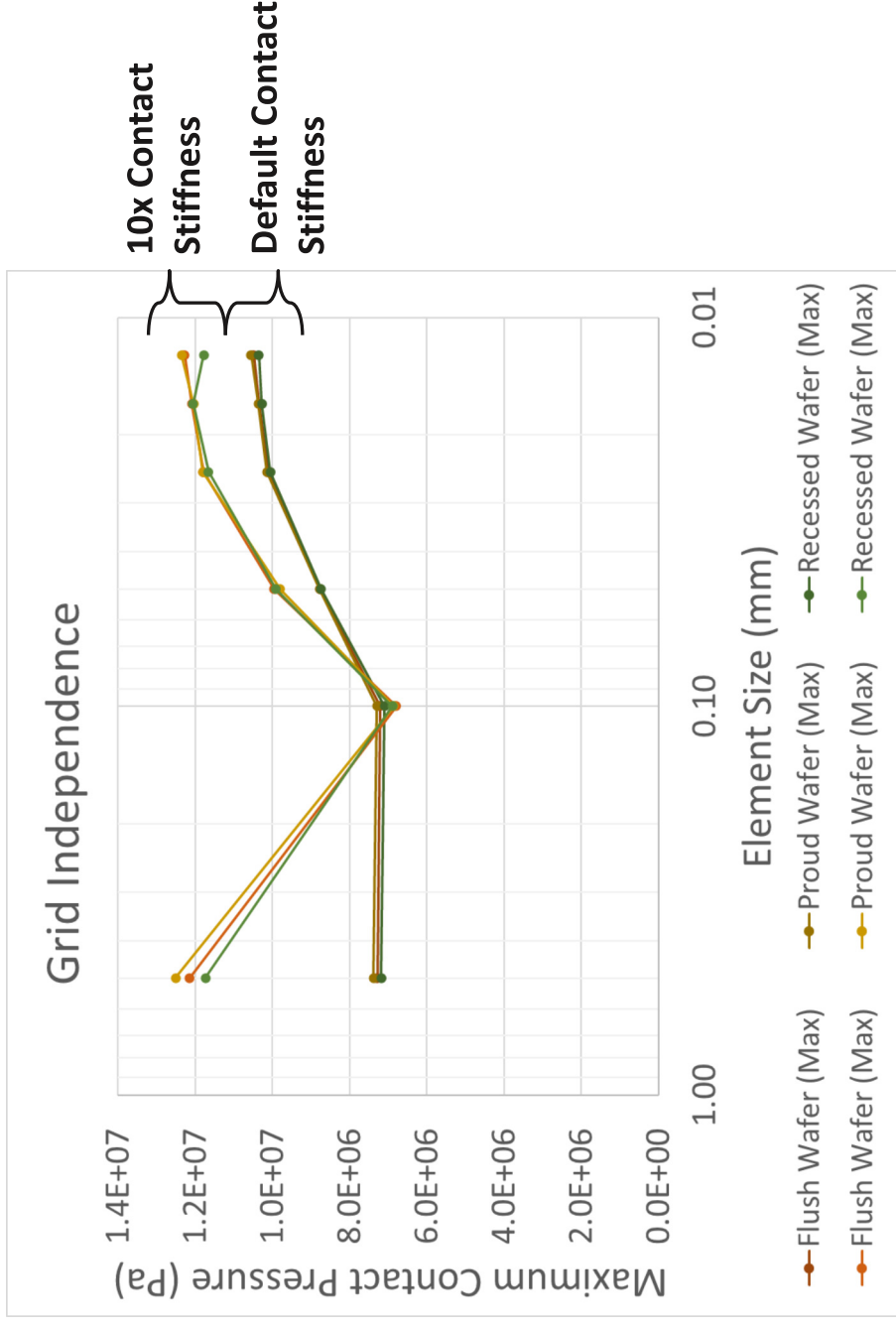
Type: Pressure
Unit: MPa
Time: 2 s



17.1% Increase in Contact Pressure

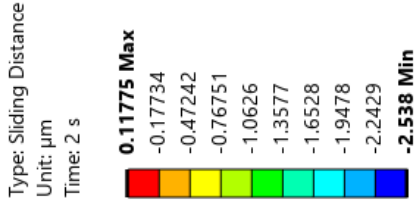
RESULTS | 2D FEM CONTACT STIFFNESS

- Contact pressure is sensitive to contact stiffness
 - Peak contact stiffness on corner of wafer
 - Contact spread influence
- Contact behavior related to contact stiffness
 - Contact surface phasing
- 0.1x contact stiffness results were too soft
 - Did not reach grid independence (not shown)
 - Excessive surface phasing

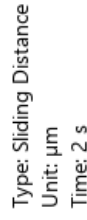


RESULTS | 2D FEM FRICTION

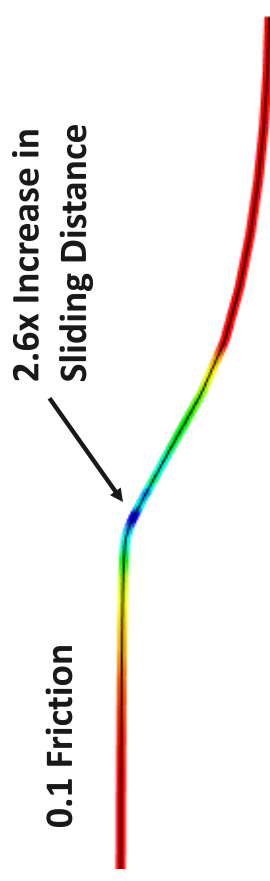
- Default contact stiffness used
- Nine model configurations tested
 - Friction: 0.1, 0.3, 0.5
 - Flush, proud and recessed wafers
- Six mesh configurations tested
 - 500 – 12.5 μm
- Contact pressure measured
- Sliding distance measured
- Deformations observed



0.3 Friction



0.1 Friction



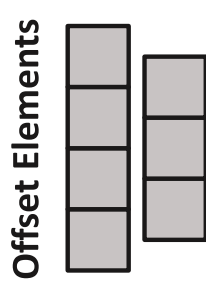
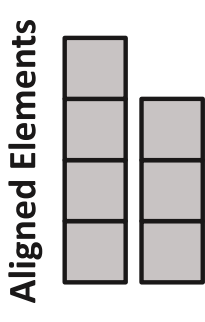
RESULTS | 2D FEM FRICTION

- Contact pressure in axisymmetric models not sensitive to friction
 - Vertical loads only
 - No rotation simulated
 - Frictionless and low friction (0.1) tend to underpredict contact pressure
- Similar results for all geometry configurations
- A true wear model with rotations against pad simulated will increase the sensitivity of contact pressure to friction

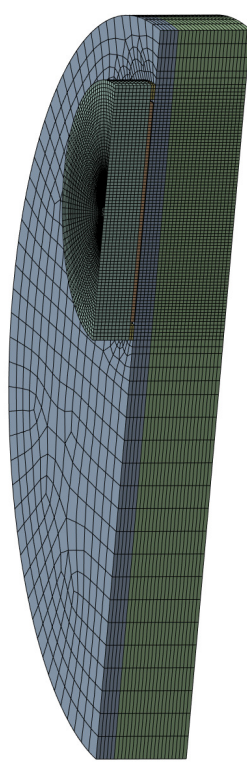


RESULTS | OTHER CONSIDERATIONS

- **Nodal Alignment**
 - The alignment of nodes from the contact surface to the target surface requires careful consideration. Best practices vary by software and contact detection methods. Refer to the software’s technical manual for advice
- **Moving from 2D to 3D**
 - 2D models are great for rapid iterations (optimization) and tuning model parameters (sensitivity studies)
 - 3D elements recommended for confirmation of final results
 - Axisymmetric and 2D plane models have reduced degrees of freedom and different element shape functions
 - A 3D model with equivalent mesh resolution will have a significant increase in computational requirements
 - Axisymmetric models in this study: a few minutes of solve time on local hardware
 - 4 cores of Intel(R) Core(TM) i9-10885H CPU @ 2.40GHz
 - 64 GB RAM
 - 3D models used in this study: several hours on solver rack
 - 12 cores of Intel(R) Core(TM) i9-7920X CPU @ 2.9GHz
 - 128 GB RAM



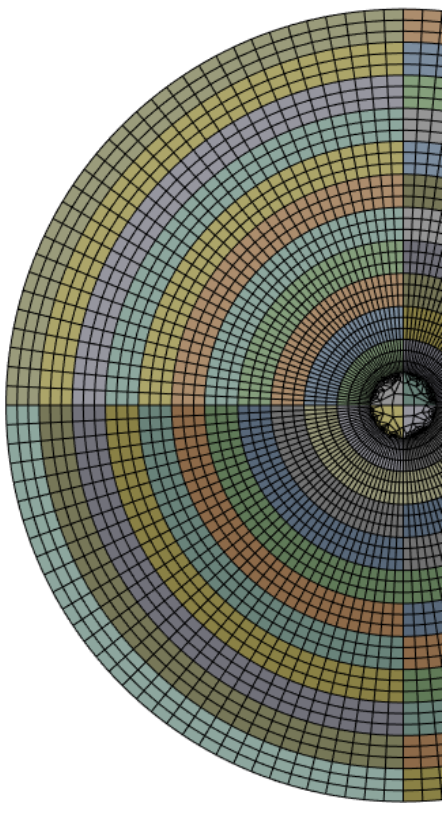
3D FEM of Single-Sided CMP Assembly



CONCLUSIONS AND FUTURE WORK

- Contact pressure, the performance-driving output of CMP simulations, is sensitive to the methodology and parameters used to define the contact between the wafer and the polishing pads
 - Contact stiffness is the leading driver of sensitivity to results
 - Contact stiffness should be tuned to avoid contact surface phasing
 - while still maintaining reasonable convergence times
 - It is important to consult the software's technical manual to ensure compliance with best practices
- During design and sensitivity studies, use model reducing strategies (2D, axisymmetric, symmetry)
 - Maintain reasonable grid independence (mesh size convergence)
- Future work to be performed
 - Optimization routines recommendation
 - Tuning wafer/retainer heights
 - Multi-zone secondary pressure carrier designs

Multizone FEM



quartus

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CITATIONS

[1] J. Chen, Y. Liu, D. Wang, W. Yu, and L. Zhu, "Numerical Analysis in Double-Sided Polishing: Mechanism Exploration of Edge Roll-Off," *Materials*, vol. 17, no. 19, p. 4761, Sep. 2024, doi: <https://doi.org/10.3390/ma17194761>.